## 3D Thermal Modelling and Simulation of Ti-6Al-4V alloy Processed by Selective Laser Melting

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## Abstract

Selective laser melting (SLM) is the most widely used metal additive manufacturing (AM) technique due to its ability to manufacture complex-shaped parts with the desired tolerances. In the present study, a three-dimensional finite element (FE) heat transfer model for the SLM process was developed and multi-track simulations were conducted to predict maximum temperatures and melt pool dimensions depends on the process parameters such as laser power, scanning speed and hatching distance for Ti6Al4V powder. FE simulations for different process parameters were conducted in ABAQUS as it provides a parametric job possibility within its Fortran subroutines. Goldak volumetric laser heat source model was used as the flux source and material properties were revealed as temperature-dependent. During laser scanning, it was observed that powder material is melted by heat source through at least one layer and this leads to interlayer connection as expected. Besides, maximum temperatures were found stable from the beginning to the end of the track until it achieves an equilibrium regime in view of temperatures with both experimental and simulation studies in the literature. Furthermore, the melt pool depth to width ratios were examined comparing to the literature. Maximum track temperatures rises from 1<sup>st</sup> track to the last 4<sup>th</sup> track as in 2550-3250 °C. On the other hand, selected process parameters were pointed out as suitable to fuse laser tracks effectively with its smaller D/W ratio than 0.5 which is possible keyhole limit.

Keywords: Heat transfer modeling, Process simulation, Selective laser melting.

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## **1. Introduction**

Additive manufacturing (AM) and specifically selective laser melting (SLM) method have become a common manufacturing method in parallel to the 4th digital revolution all over the world. SLM has serious advantages such as manufacturing ability for complex geometries without a cutting tool, using the possibility of improved computer and laser technology, less material cost, lattice structure manufacturing skills, etc. [1,2]. Although these advantages, control of the big temperature gradients, part distortions during manufacturing process sourced by heating-cooling cycles, residual stresses, keyhole case, balling effect and porosity issues bring the optimization of process parameters problem and this makes the SLM still a complex method [3-5].

In recent years, several studies have been conducted to examine and minimize the mentioned problematic issues in the SLM process. At this point, the most common approaches used in solving these problems are substrate preheating and remelting [6-8], developing scanning strategies [9-10] and experimental trials to optimize the effects of the process parameters [11-13]. Trial and error-based experimental studies to prevent or minimize the aforementioned problems can take the literature to the next step but are still time-consuming and costly [14].

In parallel to improving computer skills, modeling and simulation of the whole SLM process have become very popular in the last 20 years to predict the magnitude of heat transfer and its effects on other properties. Williams and Deckard [15] described one dimensional (1D) simulation model to investigate the effects of process parameters on density and mechanical properties of polycarbonate. Next studies continued with laser process simulations for two-dimensional (2D) and three-dimensional (3D) problems [16,9]. In another study, Ibraheem et al. [17] conducted a 3D simulation study by using the new "Element Birth and Death" method in ANSYS. The model was designed in 20x20x9 mm and they assumed that there is no latent heat effect because of small melt pools. Roberts et al. [18] conducted a simulation of SLM by using the same method to compare simulation results with Fisher's experimental data[19]. The study was covered by a 4layer ANSYS simulation with each of has 5 reciprocating laser tracks with Gaussian flux distribution. For modeling, conductivity, density and specific heat temperature-dependent values were used from the literature. Optimum process parameters belong to EOS

M250ex were used to predict the temperature of 1x1x0.15 mm powder bed and 3x3x3 mm substrate in Ref. [19]. Besides, Li et al. [20] developed a 3D FE model with a 3x1.8x0.4 mm substrate and 2.2x1.1x0.1 mm powder layer. Temperature-dependent material properties were taken from the ANSYS library. Powder bed porosity was set to 0.5 and %35 absorptivity value used in Ref. [20]. Fu and Guo [21] used Gaussian surface heat flux as a heat source and simulated temperature gradients in 4x1x0.5 mm substrate and 2x0.2x0.15 mm powder layer by using 1x10x5 micron elements in ABAQUS. They wrote a DFLUX user subroutine to move the heat source onto 5 powder layers. Temperaturedependent material properties were taken from the literature for Ti-6Al-4V alloy. Several studies also used Goldak volumetric heat source and similar material properties [22, 23]. A brief literature review on the FE process simulation of SLM for different materials can be found in Refs. [24-26].

This study focused on the FE thermal modeling of the SLM process for Ti-6Al-4V. Altough there are many thermal modelling study in the literature, most of the studies are not compherensive and effect of process parameters and related simulating datas still have blanks for this area. In this study, process model is developed using commercial software ABAQUS since it has advantages and multifunctionality by its Phytonbased coding algorithm and Fortran-based subroutines to set process parameters easily [27,28]. To obtain more accurate temperature distribution and melt pool dimensions, a detailed process model was developed firstly. Then, detailed temperature-dependent material properties and heat transfer coefficient were calculated and defined in the model. Temperature variations with their peak values were revealed for different laser power, speed and hatch distances. Finally, melt pool dimensions calculated to investigate both depth to width  $\left(\frac{D}{W}\right)$  ratios and overlap depths to understand how melt pool dimensions change for a set of effective process parameters in a certain interval.

## 2. Material and methods

In this section, FE thermal process model and its components (e.g., governing equations, heat source model, material properties, model domain, mesh and boundary conditions) were discussed in detail.

# 2.1. Governing Equations and Goldak Double Ellipsoidal Heat Source Model

In SLM, there is a transient thermal moving heat source problem exists. In the beginning, the first law of thermodynamics in view of heat flux balance can lead to the solution. Due to the powder bed can assume as a uniform distributed control volume, laser heat flux is absorbed by the powder bed and it causes an enthalpy change. On the other hand, it diffuses by conduction, convection and radiation to the environment fluid (Argon for Ti-6Al-4V). Additionally, the phase change and latent heat effect should be also considered. Powder bed also can be modeled as continuous and homogeny media. Thus, the governing equation can be written as [18,29,30];

$$\rho(T). C_p(T). \frac{\partial T}{\partial t} = k(T). \left(\frac{\partial^2 T}{x^2} + \frac{\partial^2 T}{y^2} + \frac{\partial^2 T}{z^2}\right) + q_{la} + q_{rad} + q_{conv}$$
(1)

where,  $\rho$  is density,  $C_p$  is the specific heat at constant pressure, k is thermal conductivity and derivative terms represent the Fourier 3-D conduction. While the righthand side contains heat flux that diffuses by conduction, convection, radiation and related laser moving heat source respectively, left-hand side represents stored energy or enthalpy change in powder layer. For the heat source qla, a point-related volumetric heat source called Goldak volumetric double ellipsoidal heat source described firstly by Goldak et al. [31] can be used. According to this model, moving heat source combines different front and rear parts of the heat dispersions and there is a Gaussian distribution. The literature revealed that the Goldak model is more adaptable for high energy density values [23]. The Goldak double ellipsoidal model can be defined as [31];

$$\begin{cases} \dot{Q}_{f(x,y,z)} = \frac{6.\sqrt{3}.f_{f}.\eta_{abs}.P}{\pi.\sqrt{\pi}.a_{f}.b.c} e^{-\left(\frac{3x^{2}}{a_{f}^{2}} + \frac{3y^{2}}{b^{2}} + \frac{3z^{2}}{c^{2}}\right)} \\ \dot{Q}_{r(x,y,z)} = \frac{6.\sqrt{3}.f_{r}.\eta_{abs}.P}{\pi.\sqrt{\pi}.a_{r}.b.c} e^{-\left(\frac{3x^{2}}{a_{r}^{2}} + \frac{3y^{2}}{b^{2}} + \frac{3z^{2}}{c^{2}}\right)} \end{cases}$$
(2)

In Eq. 2, *P* is laser power,  $\eta_{abs}$  is material absorbtivity,  $f_f$  and  $f_r$  are respectively front and rear portion of heat fractions which dispersed over the ellipse, a, b and c are double ellipsoidal semi-axial dimensions along x-, y and z- coordinates. Gaussian double ellipsoidal volumetric heat source was shown in Fig. 1 with elliptic parameters [32]. To apply Goldak volumetric model, a, b and c parameters must be adjusted. Since there is not enough experimental data,  $a_f$  and *b* parameters were set to equal half of laser spot diameter as 40 µm[33]. For c parameter, it was revealed that the maximum penetration depth is 65 µm for Ti6Al4V material [34]. Hence, c was selected as 65 µm. In order to calculate  $a_f$ value, the approximation below was used [31];

$$\frac{f_{\rm f}}{f_{\rm r}} = \frac{a_{\rm f}}{a_{\rm r}} \tag{3}$$

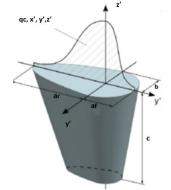


Fig 1. Goldak Double Ellipsoidal Heat Source Parameters [32].

Besides, other assumptions were also made in the modeling. At this point, convection and radiation were

applied on the top surface of the powder bed, and other surfaces were fixed at room temperature. Besides, volume shrinkage was also neglected [35,36].

#### 2.2. Material Properties

To create an accurate thermal model for SLM process, material properties found out as temperaturedependent. Solid Ti-6Al-4V properties for density, conductivity and specific heat from room temperature to melting point directly acquired from experimental data [38]. To calculate powder density, a mushy zone between solidus and liquidous temperatures is taken into account by Eqs. 4 and 5 [30];

$$\rho_{\rm po}(T) = \rho_{\rm fl}(T)\phi + (1 - \phi)\rho_{\rm so}(T) \tag{4}$$

$$\rho_{po} = \frac{\rho_{so}(T_{l})po(T_{s}) - \rho_{fl}(T_{s})}{(T_{l} - T_{s})} \cdot (T - T_{s}) + \rho_{po}(T_{s}) + \rho_{fl}(T_{s}), \ T_{s} < T < T_{l}$$

$$\rho_{po} = \frac{\Gamma_{so}(T_{l})po(T_{s}) - \rho_{fl}(T_{s})}{(T_{l} - T_{s})} \cdot (T - T_{s}) + \rho_{po}(T_{s}) + \rho_{fl}(T_{s}), \ T_{s} < T < T_{l}$$
(5)

In Eqs. 4 and 5,  $\rho$  is density and  $\phi$  is the porosity of powder bed. Sub-indices po, so and *fl* represent the powder, solid, fluid properties, respectively. Besides, *l* and *s* denote solidus and liquidous temperatures, respectively. The porosity was taken between 0.3-0.4 the most of the studies in the literature [*29,35,38,39*]. Similarly, porosity was chosen as 0.3 in this study.

During the liquid to solid pase changing process, there will be emitted an extra energy which called latent heat and raise specific heat to the higher values. To consider phase change, Eqs. 6-8 applied which has also latent heat effect to obtain specific heat during also phase changing [40,41];

$$C_{po} = ((1 - \phi)\rho_{so}C_{so} + \phi.\rho_{fl}C_{fl})/\rho_{po}$$
(6)

$$\begin{cases} cp_{po} = \begin{cases} cp_{po}(T) , T_0 \le T \le T_s \\ (\frac{cp_{so}(T_1) - cp_{po}(T_s))}{(T_1 - T_s)} . (T - T_s) + cp_{po}(T_s) , T_s < T < T_1 \\ cp_{co}, T \ge T_1 \end{cases}$$
(7)

$$c = c_{s} + \frac{L_{f}(\{T-T_{s}\})}{\Delta T_{f}^{2}} + \frac{L_{v}(\{T-T_{i}\})}{\Delta T_{v}^{2}}$$
(8)

where c is the specific heat;  $L_f$  and  $L_v$  are fusion and vaporization latent heats;  $\Delta T_f$  and  $\Delta T_v$  are fusion and vaporization temperatures differences, respectively.

Similar to specific heat and density calculations, the mushy zone was also taken into account by Eq. 11 to obtain powder conductivity values based on Eqs. 9 and 10[36];

$$\frac{\kappa_{po}}{\kappa_{fl}} = 1 - \sqrt{1 - \phi} \left( 1 + \phi. \frac{\kappa_{r}}{\kappa_{fl}} \right) + \sqrt{1 - \phi}. \left( \frac{2}{1 - \frac{\kappa_{f}}{\kappa_{so}}} \right) \cdot \left( \frac{1}{1 - \frac{\kappa_{fl}}{\kappa_{so}}} \right) \cdot \left( 1 - \frac{\kappa_{fl}}{\kappa_{so}} \right) \cdot \ln \left( \frac{\kappa_{so}}{\kappa_{fl}} \right) + \left( \frac{\kappa_{r}}{\kappa_{fl}} \right)$$
(9)

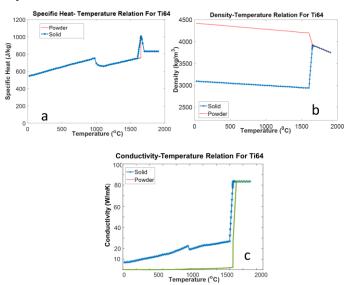
$$K_r = 4. F. \sigma. \left(T_p\right)^3. D_p \tag{10}$$

$$k_{eff} = \begin{cases} k_{po}, T_0 \leq T \leq T_s \\ ((k_{so}(T_l) - k_{po}(T_s))/(T_l - T_s)) + k_{po}(T_s), T_s < T < T_l \\ k_{so}, T \geq T_l \end{cases}$$
(11)

where  $K_r$  is a constant which is for micro radiation effects, F is an experimental constant,  $D_p$  is average powder diameter. Fluid density is calculated for Argon as temperature-dependent, fluid specific heat used its constant value between 0 to 2000 °C for Argon, and

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powder bed conductivity was calculated as temperature-dependent. All calculated material properties were showed in Fig. 2 as graphical representations.



**Fig 2**. Solid and Powder Material Properties for Ti-6Al-4V a) Specific Heat, b) Density and c) Conductivity.

Both natural and forced convective terms were considered in literature [34,38]. We assumed the natural one is acceptable since EOS Model M280 SLM machine contains an inert gas input mouth about 10 cm above the powder bed. Thus, convection type can be assumed as natural convection due to the distance between gas flow and powder bed top surface in view of the model dimensions. To apply natural convection and find out convection coefficient, definitions in Eq. 12 and the solution formula in Eq. 13 can be used;

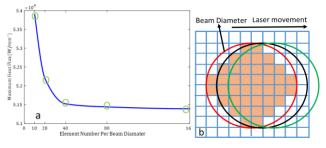
$$h_{c} = \frac{Nuk_{f}}{L}, \quad Gr = \frac{g\left(L^{3}\rho_{f}^{2}\beta_{f}(T-T_{inf})\right)}{\eta_{f}^{2}}, \quad Pr = \frac{\eta_{f}C_{f}}{k_{f}}$$
(12)

$$\sqrt{\mathrm{Nu}} = \sqrt{\mathrm{Nu}_{0}} + \left(\frac{\frac{\mathrm{GrPr}}{300}}{\left(1 + \left(\frac{0.5}{\mathrm{Pr}}\right)^{\frac{9}{16}}\right)^{\frac{16}{9}}}\right)^{\frac{1}{6}}, \ \mathrm{L} = \frac{\mathrm{Surface\ area}}{\mathrm{Perimeter}}$$
(13)

By using Eq. 12 and 13, average  $h_c=20 \text{ W/m}^2\text{K}$  value calculated by using EES (Engineering Equation Solver). For the radiative term, the emissivity value is also a critical parameter. Some researchers have revealed the temperature-dependent emissivity values for Ti-6Al-4V [29,41]. Most of the studies were used emissivity as a constant among 0.4-0.7 [38,41]. It is known that powder emissivity has higher values than solid emissivity. Hence, in this study, the emissivity value was chosen as 0.7 for Ti-6Al-4V [41].

#### 2.3. Computational Domain, FE Model Parameters and Boundary Conditions

ABAQUS adopts an implicit backward difference scheme to calculate nodal temperatures within Newton Raphson iterative method. Consistent with the proposed method, DC3D8 8 node brick element was employed in the 3D model. To determine optimum element size, a mesh convergence test was conducted from 40x40x10  $\mu$ m to 5x5x10  $\mu$ m with ½ ratio increments. Finally, element dimensions were determined as 10x10x10  $\mu$ m<sup>3</sup> to conduct an efficient analysis. Fig. 3a shows the result of different meshes at 170 W power, 1.25 m/s speed and 100  $\mu$ m hatch, and Fig 3b reveals the element number throughout the beam area.



**Fig 3**. a) Maximum Heat Flux Versus Element Number Per Beam Diameter For The Center Point of The First Track and b) Element Distribution Per Laser Beam

The computational domain was determined as  $0.9x0.9x0.4 \text{ mm}^3$  for substrate and  $0.7x0.7x0.03 \text{ mm}^3$  for the powder bed. 100 µm of offset value was defined from the edges in direction of x- and y- coordinates to avoid negative edge heat accumulation effects and to prevent heat transfer to the space at half of the edges. Fig. 4 shows the FE simulation model for deposition with three layers.

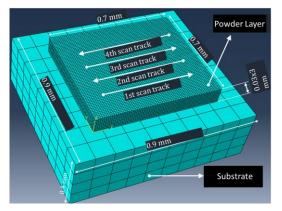


Fig. 4. 3D Simulation Model

In this study, three different power, speed and hatch values used in simulations. Other parameters were kept constant. Table 1 shows the used parameters.

Table 1. Simulation and process parameters.
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Laser Power (W)	150-170 - 190	Solidus and Liquidous (°C)	1600, 1650,
Scaning Speed (m/s)	1-1.25 - 1.5	Vaporization (°C)	2860
Hatching Distance (mm)	0.08 - 0.1 - 0.12	Latent Heat Of Fusion and Vapor (J/kg)	295000, 9860000
Beam Diam. (mm)	0.08	Porosity (%)	30
Layer	0.03	Absorptivity (%)	70

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Thick.(mm)			
Powder Material	Ti6Al4V	Emissivity (%)	70

Selecting a suitable time step is also crucial to get accurate temperature results. For this goal, step time calculated with the following equation [42];

$$\min \Delta t = \frac{\Delta x}{v} = \frac{0.01 \text{ mm}}{1250 \text{ mm/s}} = 8x10^{-6} \text{ s}$$
(14)

where  $\Delta t$  is the step time enable to select for each scanning track.  $\Delta x$  is the smallest element dimension at x- or y- coordinates since the length and width are equal for chosen mesh design with DC3D8. v is scanning speed on powder bed layer. While speed was changing, step time was calculated again and changed.

Before defining other boundary conditions, the room temperature was kept at 25 °C. Convection and radiation were assumed to be from the top surface of the powder bed to the environmental inert gas. Side faces were kept at room temperature  $T_0$ , and defined as

$$T(x, y, z, t) = T_0$$
 (15)

At the top surface, heat transfer boundary condition was defined with a temperature dependent *k*, as;

$$q_{la} = \sigma \varepsilon \left( T_{top.}^4 - T_{env.}^4 \right) + h_c \left( T - T_{env} \right) + k \left( \frac{\partial^2 T}{x^2} + \frac{\partial^2 T}{y^2} + \frac{\partial^2 T}{z^2} \right) \quad (16)$$

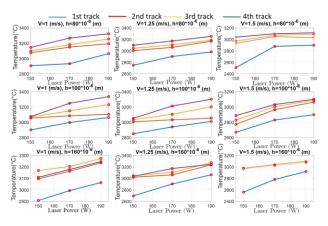
## 3. Results and discussion

#### 3.1. Temperature Distribution

Maximum calculated temperature values were plotted in Fig. 5 for different laser power values. For the 1st track, maximum temperatures are between 2850-3000 °C. For the rest of the 3<sup>rd</sup> scanning track, this interval rises to 3000-3250 °C. Patil and Yadava [43] showed that maximum simulation temperatures are about 2978 K. Kolossov [44] plotted both simulated and experimental temperatures approximately as 2533 and 2400 °C respectively. Both studies were conducted at 2 W power and 1 m/s speed. Romano et al. [45] found the maximum temperatures as 2369.57 K for 100 W with using 100 µm layer thickness. Zhuang et al. [38] predicted maximum temperature approximately as 2500 °C . Huang et al. [35] showed that the maximum temperatures occur among 2400-2600 K in 100 W-120 W interval. Soylemez [46] predicted maximum temperature limit as 3500 °C. On the other hand, maximum temperature changes about 400-600 °C at 2nd or 3<sup>rd</sup> track from 150 W to 190 W (see Fig. 5). Considering this increment in our simulation, the predicted maximum temperatures can be evaluated coherent with the literature. The effects of hatching distance and scanning speed on temperature results were also presented in Fig. 5. According to the maximal temperatures, heat is transferring from 1<sup>st</sup> track to the adjacent track faster. Because there is not exist any preheated track or zone at the beginning for 1<sup>st</sup> track. Heat transfer from 2<sup>nd</sup> to 3<sup>rd</sup> or 3<sup>rd</sup> to 4<sup>th</sup> track is much smaller and maximum temperatures are much similar to each other. On the other hand, it is seen that the

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hatch distance is much more important compared to the laser scanning speed. For a linear heat transfer between tracks, among hatch values used in this study, 120  $\mu m$  can be considered as the most suitable value.



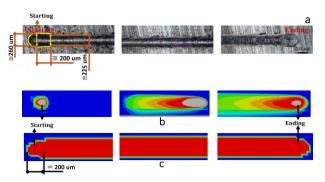
**Fig 5.** Maximum temperatures against laser power, scanning speed and hatching distance.

#### 3.2. Melt Pool Dimensions

In this study, simulations were conducted with continuously moving laser motion from the beginning to the end of a track. Thus, there is no acceleration effect at the start of tracks. Similarly, at the end of the tracks, laser power cuts off instantaneously which has again an acceleration effect at the end of a track in the manufacturing process. In practice, there are much bigger track zones because of the galvanometer mirror acceleration ramping [47]. In Fig. 6, our simulation results were compared with experimental results in literature [47]. Gong et al. [47] determined the accelerated zone at the beginning of a track approximately as 200  $\mu$ m. In the simulation, steady melt pool width occurs approximately at 200 µm far away from starting point. Based on this data, melt pools were investigated at a cross-sectional area 300 µm far away from starting point to find melt pool dimensions. For each track, maximum depth (D) and width (W) values were measured and  $\frac{D}{W}$  ratios were calculated to determine keyhole possibility. According to the Soylemez, when  $\frac{D}{W} > 0.5$ , there can be a keyhole problem, and at depth values which satisfies  $\frac{D}{W} > 1.2$ , keyhole problem is inevitable phenomena [46]. Hence, a critical depth (CD) value was defined to obtain limits of maximum efficient depth values for each track.

When predicted pool depths and widths were compared with the experimental and simulation results in the literature, it was observed that pool dimensions are similar to both experimental and simulation results. Fu and Guo [21] presented the melt pool depths depends on power in interval of approximately 40-120  $\mu$ m and widths are from 50 to 160  $\mu$ m levels within error bars in interval of 20 W to 80 W laser power. Romano et al. [45] was predicted melt pool width and depth as 240 and 120  $\mu$ m for 100  $\mu$ m layer thickness at 100 W. Soylemez [46] observed width value about 150

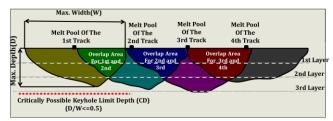
μm and depth value about 100 μm for 150 W and 1200 mm/s speed combination. In this study, for 150 W and 1.25 m/s, width and depth were predicted as 143 μm and 54 μm for the 1st track, respectively. Zhuang et al. [38] found depth about 60 μm and width about 180 μm for 120 W power using 30 μm layer thickness at the end of the 5<sup>th</sup> track. Melt pool depth and width were predicted in the study of Huang et al. [35]. For 120 W power, width and depth were obtained as 150 μm and 60 μm, respectively.



**Fig 6.** a) Single Laser Track [47], b) Simulation Melt Pool Zone and c) Solidified Zones.

Comparing to the literature, as mentioned above, our melt pool dimensions are similar to the other studies. Discrepancies between simulated melt pool dimensions are probably caused by different laser powers, scanning speeds and hatching distances or assumptions for a low cost simulation.

In Fig. 7, a schematic of simulation melt pool overlaps was shown with different colors to represent the measurement process of pool dimensions. Fig. 7 selected to explain melt pool depth and width calculation process at 190 W- 1.5 m/s – 80  $\mu$ m. The reason selection of this parameter is that the maximum  $\frac{D}{W}$  ratio was at this combination for the 1<sup>st</sup> track. Calculations done for 27 parameter combinations were given in Table 1.



**Fig 7.** Melt Pool Dimensions For Adjacent 4 Tracks At Cross Section 300  $\mu$ m Far Away From Initial Point (Parameters; Power=150 W, Speed 1 m/s, Hatch= 120  $\mu$ m].

With following Fig. 7 for each parameter combination, the biggest  $\frac{D}{W}$  ratio was found. Considering tried power, hatch and speed combinations as effective process parameters, thermal model successfully predicted the  $\frac{D}{W}$ ratios since there is no keyhole effect was found. On the other hand, it can be seen from Fig. 7 that the overlap area has enough depth to melt all powder since it rises to a depth which about at least two times powder layer thickness.

Table 2 shows CD values for all combinations and at all

tracks. It was found that melt pool width and depth

predicted as rising and  $\frac{D}{W}$  ratios as decreasing from 1<sup>st</sup> track to 4<sup>th</sup> track regardless of power, speed or hatch distance values. On the other hand, for all combinations,  $\frac{D}{W}$  ratios were found  $\leq 0.5$ 

Table 2. Melt Pool Dimensions, Dimension Ratios and Critical Depth Values For 4 Separate Tracks.

## **4.** Conclusions

In this study, a detailed 3D finite element heat transfer model for the SLM process was developed based on parameters that were used in EOS M280 SLM device and multi-track simulations were conducted to predict maximum temperatures and melt pool dimensions depends on 27 different process parameter configurations. Results are briefly discussed as follows:

• Predicted maximum temperatures and melt pool dimensions are consistent with existing literature and revealed model gives reliable results.

• According to calculated D/W ratios and CD values, critical keyhole possibility doesn't occur and laser power is more effective than speed and hatch.

• Altough simulation cannot mimic bigger widths at start/end of a track which sourced by the ramping of Galvano mirrors in real manufacturing process, it doesn't cause a problem to predict maximum temperatures or melting pools accurately.

Future study will be on the calibration of the model predictions with the help of experimental multitrack specimen manufacturings. Cross sectional area of EDM machined specimen tracks will be investigated to compare with simulation results. By the way, in-situ temperature measurements by using thermocouple and/or ratio type pyrometer is also planned to provide an experimental data set for maximum temperatures and cooling rates.

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#### Author's statement

Conflict of interest: Authors state no conflict of interest.

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